

SKMM 3033

Finite Element Method

Topic 7: Prismatic beam elements

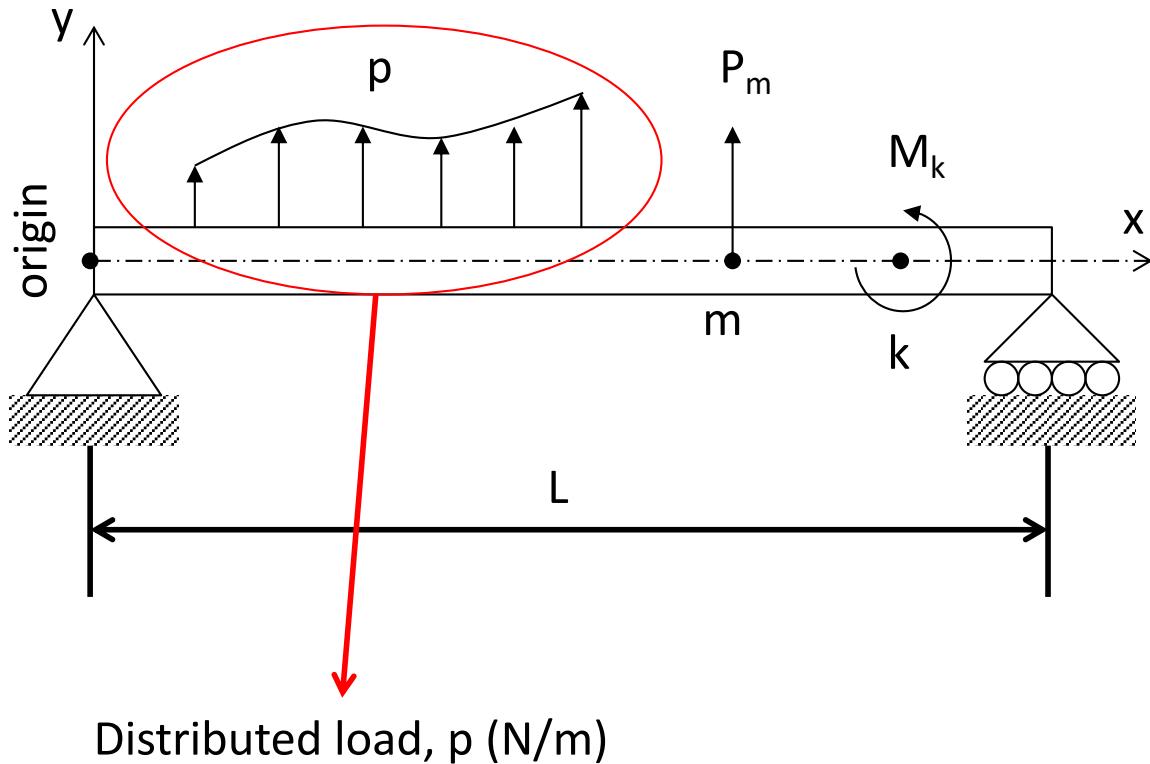
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Engineering
Faculty of Mechanical Engineering

By the end of the notes:

The students are expected:

- To assemble the stiffness matrix and load vector for a beam element
- To assemble the global stiffness matrix for a beam structure

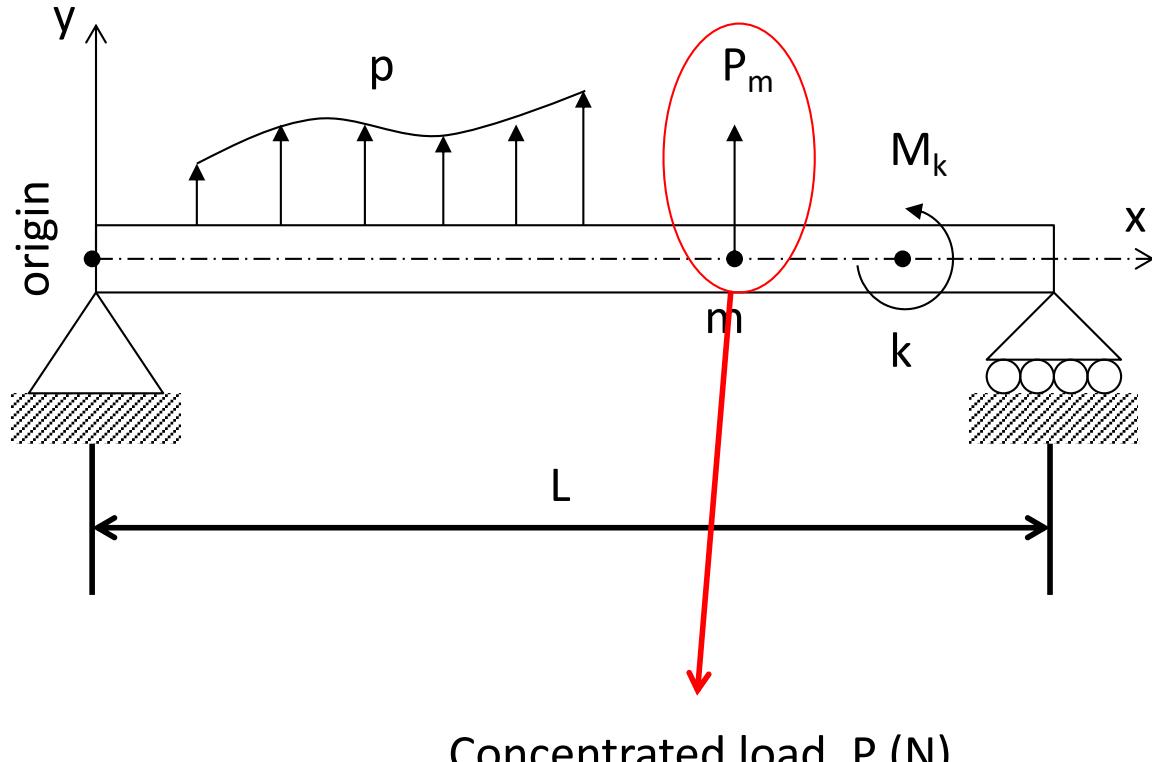
Beam Analysis



In FEM, we are interested mainly in:

1. **Loading condition**
2. Displacement, q
3. Stiffness matrix, k

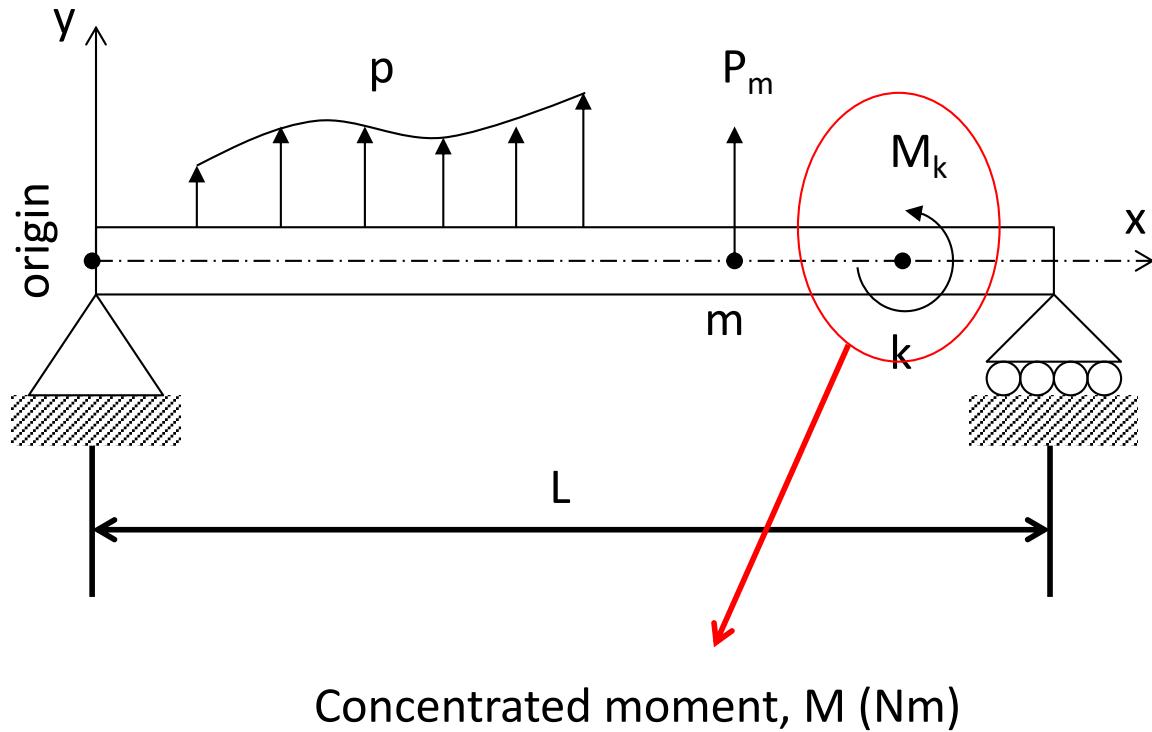
Beam Analysis



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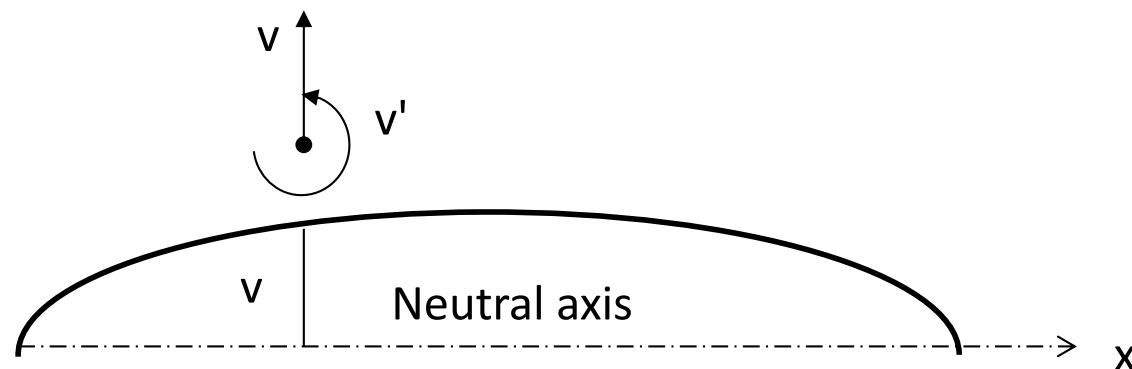
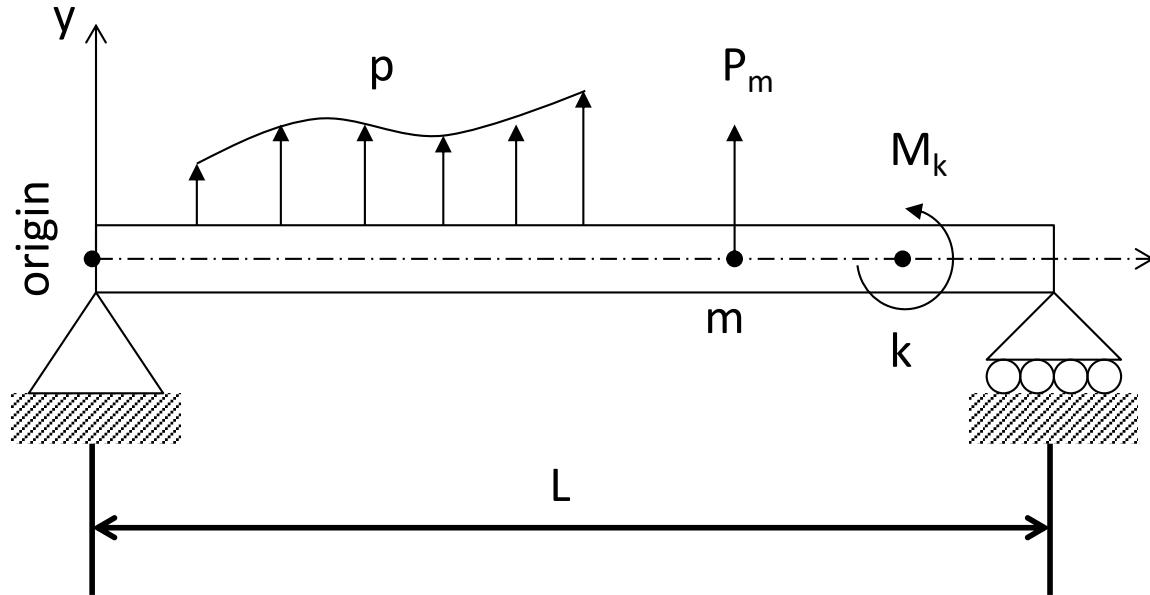
Beam Analysis



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Beam Analysis



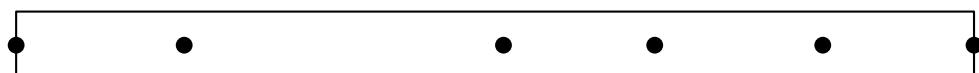
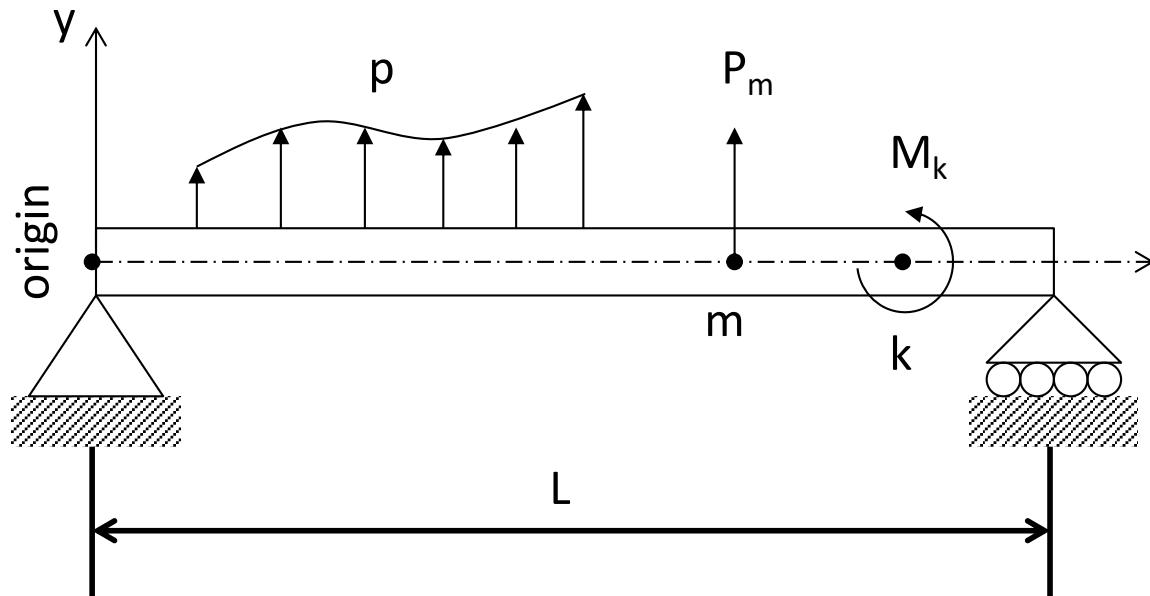
In FEM, we are interested mainly in:

1. Loading condition
2. **Displacement, q**
3. Stiffness matrix, k

Deformation along neutral axis:

v =vertical deflection
 v' =rotation or slope

FEA - Beams



1

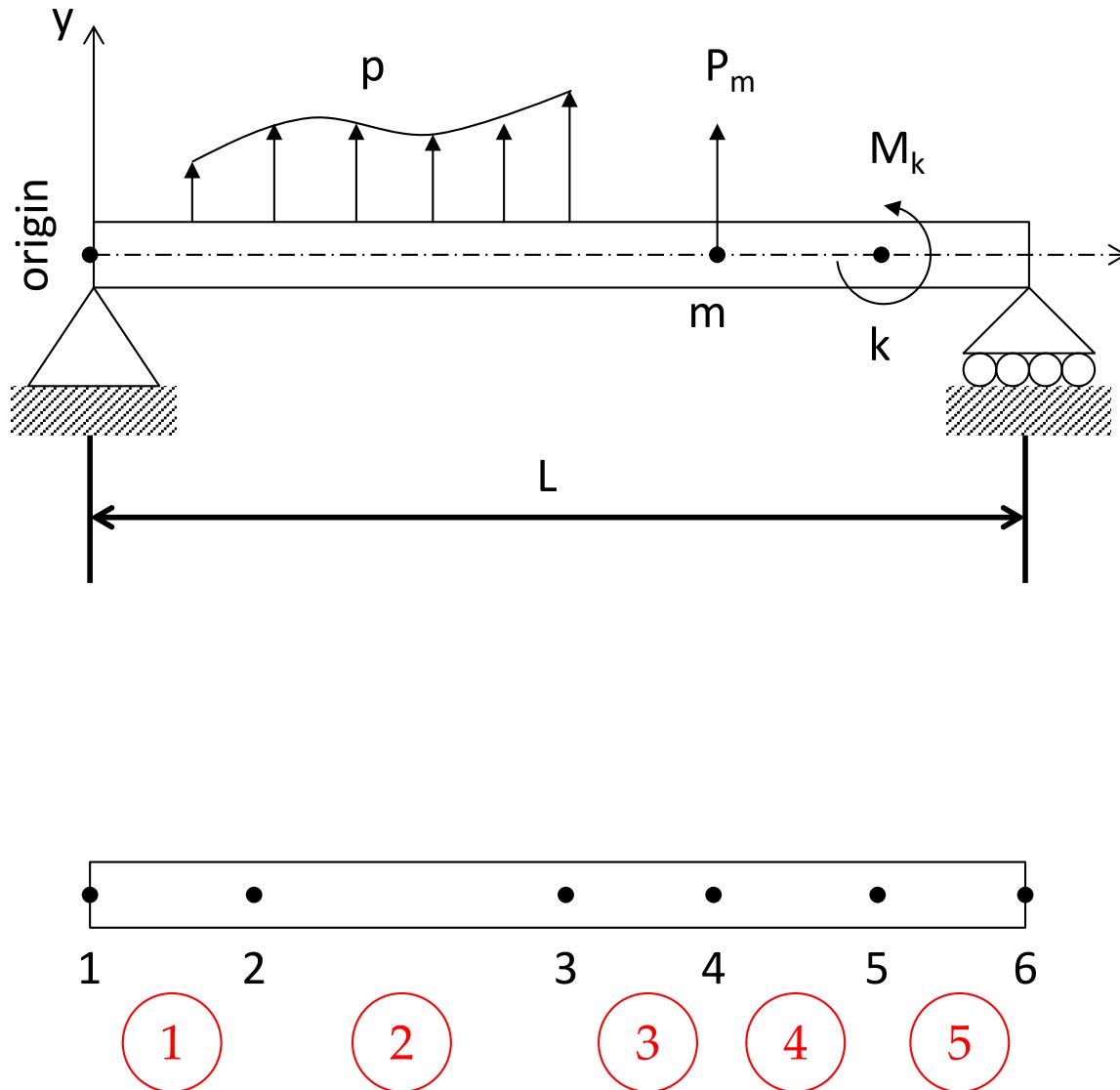
2

3

4

5

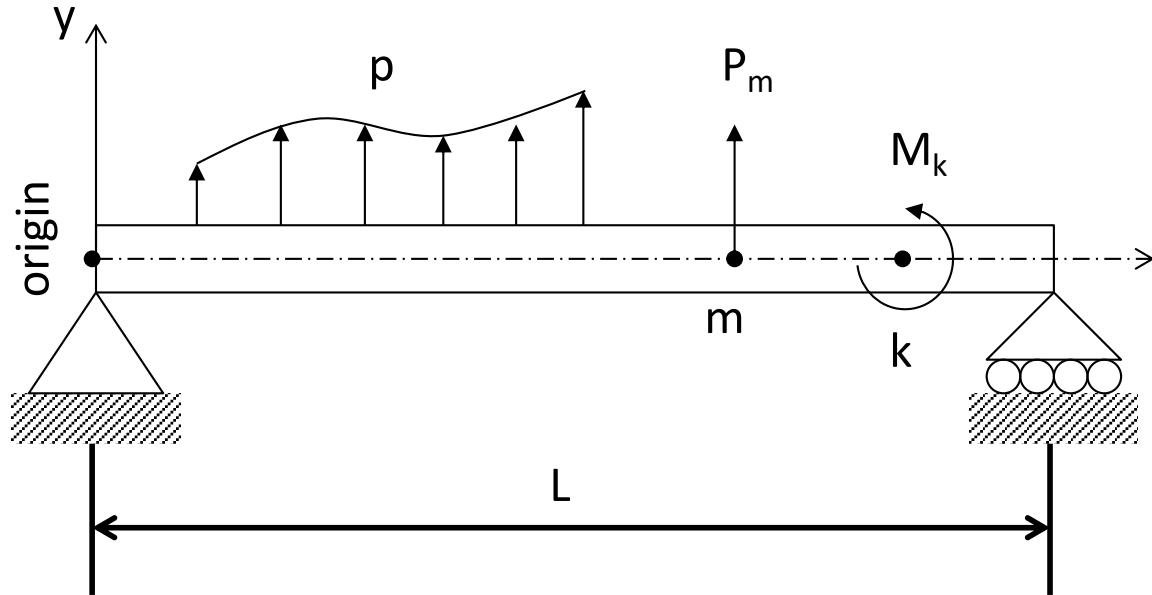
FEA - Beams



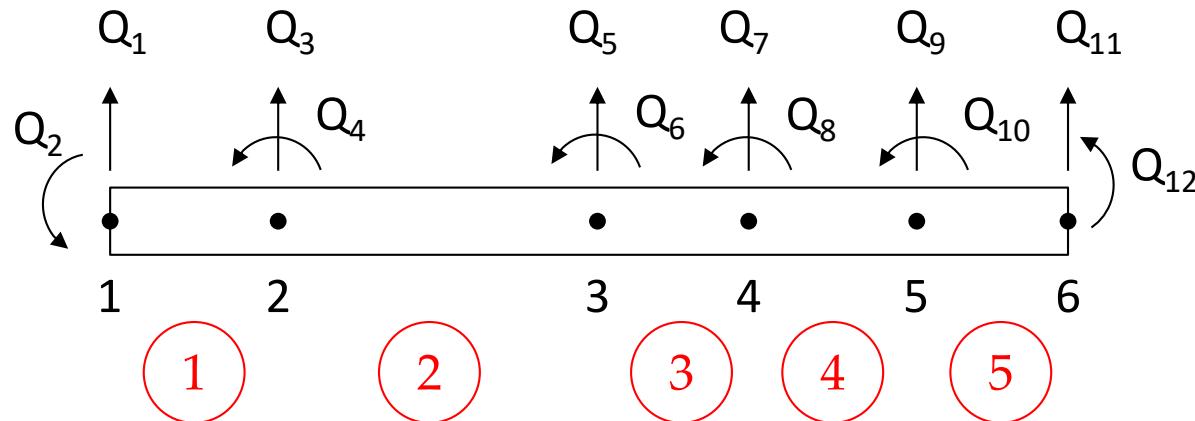
Element Connectivity Table

Element	Nodes	
1	1	2
2	2	3
3	3	4
4	4	5
5	5	6

FEA - Beams

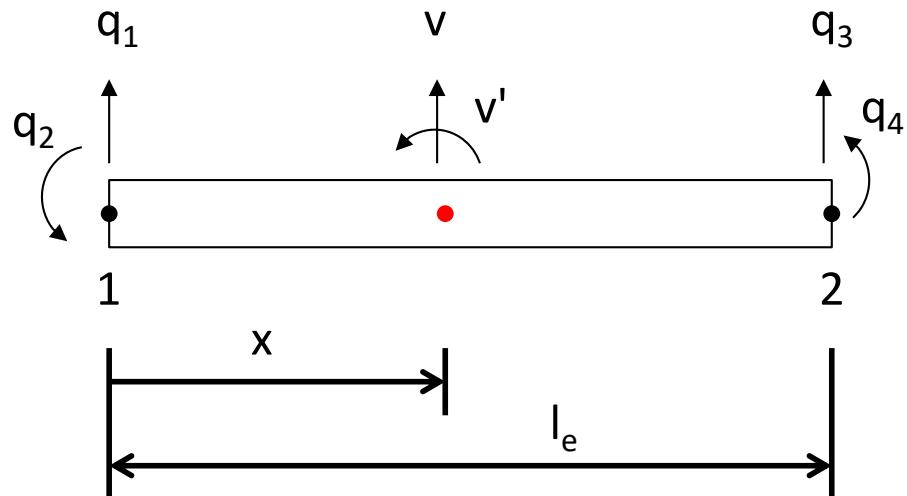


- Each node has 2 degrees of freedom
- One is the transverse displacement
- The other is the slope or rotation



Element Analysis (Beam element)

Beam Element



Natural Coordinate:

For a 2-noded beam element, we assume:

$$x_1 \rightarrow \xi = -1$$

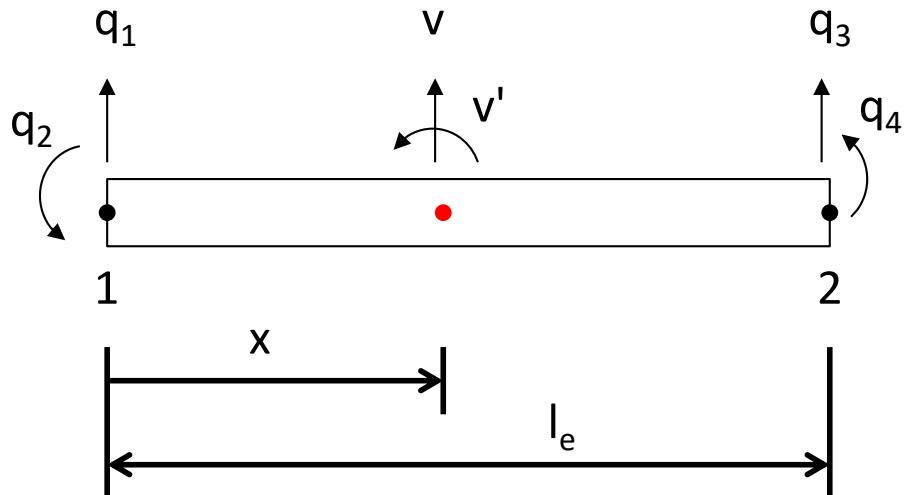
$$x_2 \rightarrow \xi = +1$$

$$\therefore \xi = 2\left(\frac{x}{l_e}\right) - 1$$

Note: All directions shown produces positive deformations

Similar to 1-D bar element

Beam Element



Also known as
Hermite Shape
Functions

$$H_1 = \left(\frac{\xi^3 - 3\xi + 2}{4} \right)$$

$$H_2 = \left(\frac{\xi^3 - \xi^2 - \xi + 1}{4} \right)$$

$$H_3 = \left(\frac{-\xi^3 + 3\xi + 2}{4} \right)$$

$$H_4 = \left(\frac{\xi^3 + \xi^2 - \xi - 1}{4} \right)$$

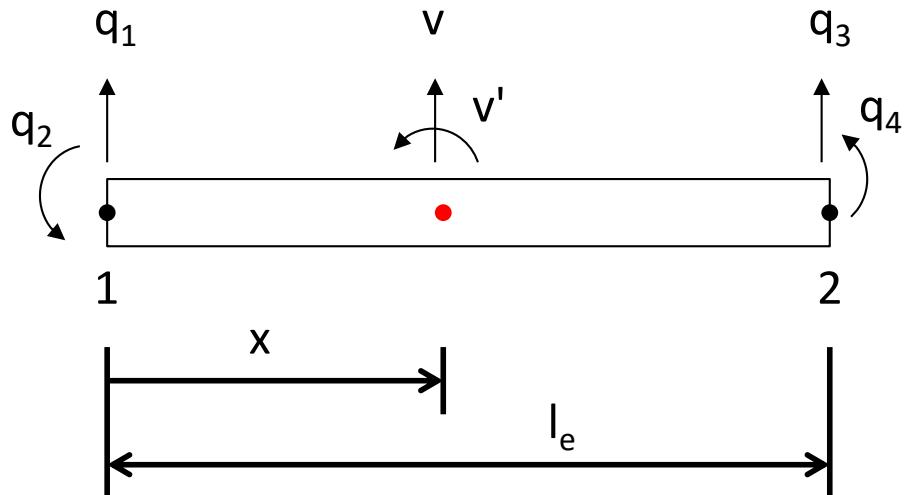
Shape Functions:

- Beam element has 4 degrees of freedom
- Appropriate to use cubic function to describe displacement field

Displacement field:

$$v = H_1 q_1 + \frac{l_e}{2} H_2 q_2 + H_3 q_3 + \frac{l_e}{2} H_4 q_4$$

Beam Element



Slope:

$$v' = \frac{2}{l_e} \left(\frac{3\xi^2 - 3}{4} \right) q_1 + \left(\frac{3\xi^2 - 2\xi - 1}{4} \right) q_2 + \frac{2}{l_e} \left(\frac{-3\xi^2 + 3}{4} \right) q_3 + \left(\frac{3\xi^2 + 2\xi - 1}{4} \right) q_4$$

$$v' = \begin{pmatrix} \frac{2}{l_e} \left(\frac{3\xi^2 - 3}{4} \right) & \left(\frac{3\xi^2 - 2\xi - 1}{4} \right) & \frac{2}{l_e} \left(\frac{-3\xi^2 + 3}{4} \right) & \left(\frac{3\xi^2 + 2\xi - 1}{4} \right) \end{pmatrix} \begin{pmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{pmatrix}$$

Shape Functions:

- Beam element has 4 degrees of freedom
- Appropriate to use cubic function to describe displacement field

Beam Element

Potential Energy Approach:

$$\Pi = U + \Omega$$

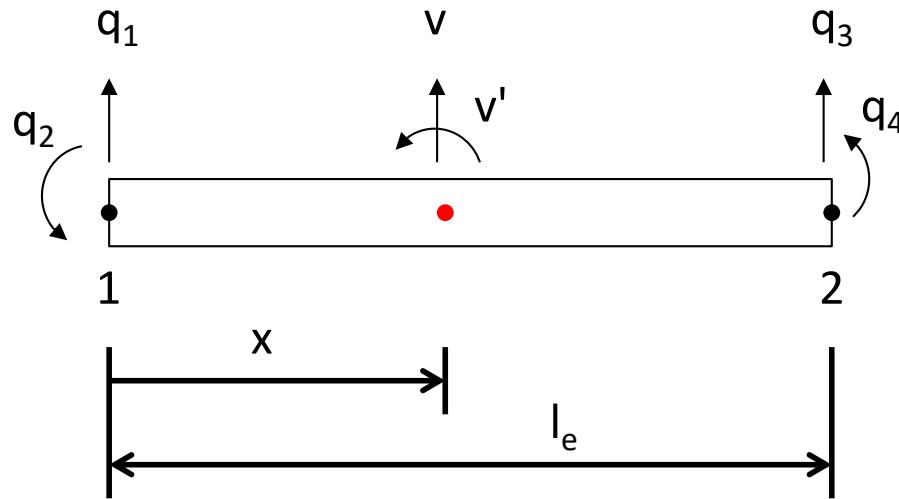
Strain Energy

Work Potential

For beam
element

$$\Pi = \frac{1}{2} EI \int_L \left(\frac{d^2 v}{dx^2} \right)^2 dx - \int_L p v dx - \sum_m P_m v_m - \sum_k M_k v'_k$$

Beam Element



Element Stiffness Matrix, k

For beam element, the element strain energy is given as:

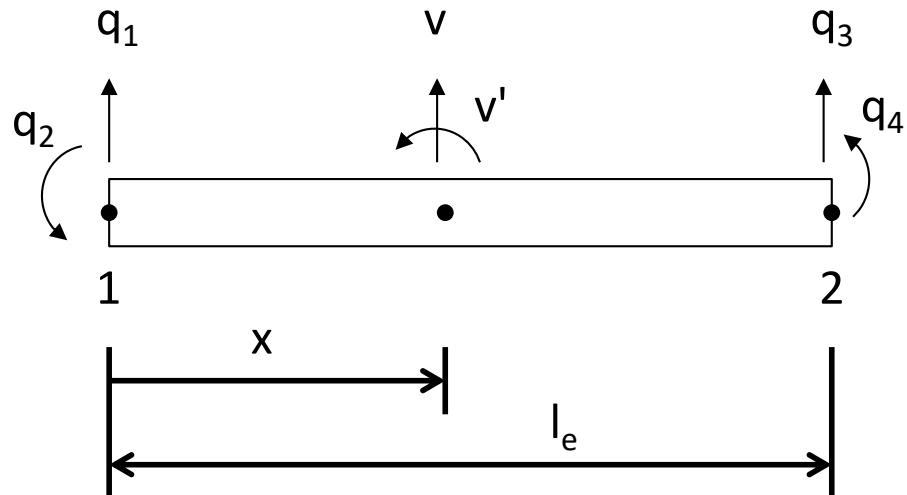
$$U_e = \frac{1}{2} EI \int \left(\frac{d^2 v}{dx^2} \right)^2 dx$$

$$\frac{d^2 v}{dx^2} = \frac{d}{d\xi} \cdot \frac{dv}{dx} \cdot \frac{d\xi}{dx}$$

$$\xi = 2 \left(\frac{x}{l_e} \right) - 1$$

$$\therefore \frac{d\xi}{dx} = \frac{2}{l_e}$$

Beam Element



Element Stiffness Matrix, k

For beam element, the element strain energy is given as:

$$U_e = \frac{1}{2} EI \int \left(\frac{d^2 v}{dx_2} \right)^2 dx$$

$$\frac{d^2 v}{dx_2} = \frac{1}{l_e^2} \begin{pmatrix} 6\xi & l_e(3\xi - 1) & -6\xi & l_e(3\xi + 1) \end{pmatrix} \begin{pmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{pmatrix}$$

$$\therefore \frac{d^2 v}{dx_2} = \frac{1}{l_e^2} \{B\} \{q\}$$

Beam Element

$$U_e = \frac{1}{2} EI \int \left(\frac{1}{l_e^2} \{B\} \{q\} \right)^2 dx$$

$$U_e = \frac{1}{2} EI \int \left(\frac{1}{l_e^2} \{B\} \{q\} \right)^2 \frac{l_e}{2} d\xi$$

$$U_e = \frac{1}{2} EI \int \frac{1}{2l_e^3} \{q\}^T \{B\}^T \{B\} \{q\} d\xi$$

$$U_e = \frac{1}{2} \{q\}^T EI \int \frac{1}{2l_e^3} \{B\}^T \{B\} d\xi \{q\}$$

Element Stiffness Matrix, k

For beam element, the element strain energy is given as:

$$U_e = \frac{1}{2} EI \int \left(\frac{d^2 v}{dx_2} \right)^2 dx$$

$$\xi = 2 \left(\frac{x}{l_e} \right) - 1$$

$$\frac{d\xi}{dx} = \frac{2}{l_e} \rightarrow \therefore dx = \frac{l_e}{2} d\xi$$

Beam Element

$$U_e = \frac{1}{2} EI \int \left(\frac{1}{l_e^2} \{B\} \{q\} \right)^2 dx$$

$$U_e = \frac{1}{2} EI \int \left(\frac{1}{l_e^2} \{B\} \{q\} \right)^2 \frac{l_e}{2} d\xi$$

$$U_e = \frac{1}{2} EI \int \frac{1}{2l_e^3} \{q\}^T \{B\}^T \{B\} \{q\} d\xi$$

$$U_e = \frac{1}{2} \{q\}^T EI \boxed{\int \frac{1}{2l_e^3} \{B\}^T \{B\} d\xi} \{q\}$$

Element Stiffness Matrix, k

For beam element, the element strain energy is given as:

$$U_e = \frac{1}{2} EI \int \left(\frac{d^2 v}{dx_2} \right)^2 dx$$

$$\{B\} \{q\} = \{q\}^T \{B\}^T$$

Beam Element

$$\int_{-1}^1 \{B\}^T \{B\} d\xi$$

$$= \int_{-1}^1 \begin{pmatrix} 6\xi \\ l_e(3\xi - 1) \\ -6\xi \\ l_e(3\xi + 1) \end{pmatrix} \begin{pmatrix} 6\xi & l_e(3\xi - 1) & -6\xi & l_e(3\xi + 1) \end{pmatrix} d\xi$$

$$= \int_{-1}^1 \begin{pmatrix} 36\xi^2 & 6\xi(3\xi - 1)l_e & -36\xi^2 & 6\xi(3\xi + 1)l_e \\ 6\xi(3\xi - 1)l_e & (3\xi - 1)^2 l_e^2 & -6\xi(3\xi - 1)l_e & (3\xi - 1)(3\xi + 1)l_e^2 \\ -36\xi^2 & -6\xi(3\xi - 1)l_e & 36\xi^2 & -6\xi(3\xi + 1)l_e \\ 6\xi(3\xi + 1)l_e & (3\xi + 1)(3\xi - 1)l_e^2 & -6\xi(3\xi + 1)l_e & (3\xi + 1)^2 l_e^2 \end{pmatrix} d\xi$$

$$= \begin{pmatrix} 24 & 12l_e & -24 & 12l_e \\ 12l_e & 8l_e^2 & -12l_e & 4l_e^2 \\ -24 & -12l_e & 24 & -12l_e \\ 12l_e & 4l_e^2 & -12l_e & 8l_e^2 \end{pmatrix}$$

Beam Element

$$U_e = \frac{1}{2} \{q\}^T \frac{EI}{2l_e^3} \begin{pmatrix} 24 & 12l_e & -24 & 12l_e \\ 12l_e & 8l_e^2 & -12l_e & 4l_e^2 \\ -24 & -12l_e & 24 & -12l_e \\ 12l_e & 4l_e^2 & -12l_e & 8l_e^2 \end{pmatrix} \{q\}$$

$$\therefore U_e = \frac{1}{2} \{q\}^T k_e \{q\}$$

With

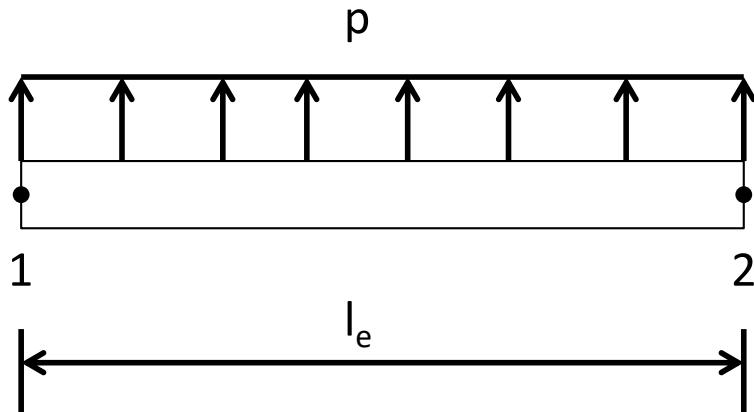
$$k_e = \frac{EI}{l_e^3} \begin{pmatrix} 12 & 6l_e & -12 & 6l_e \\ 6l_e & 4l_e^2 & -6l_e & 2l_e^2 \\ -12 & -6l_e & 12 & -6l_e \\ 6l_e & 2l_e^2 & -6l_e & 4l_e^2 \end{pmatrix}$$

Element Stiffness Matrix, k

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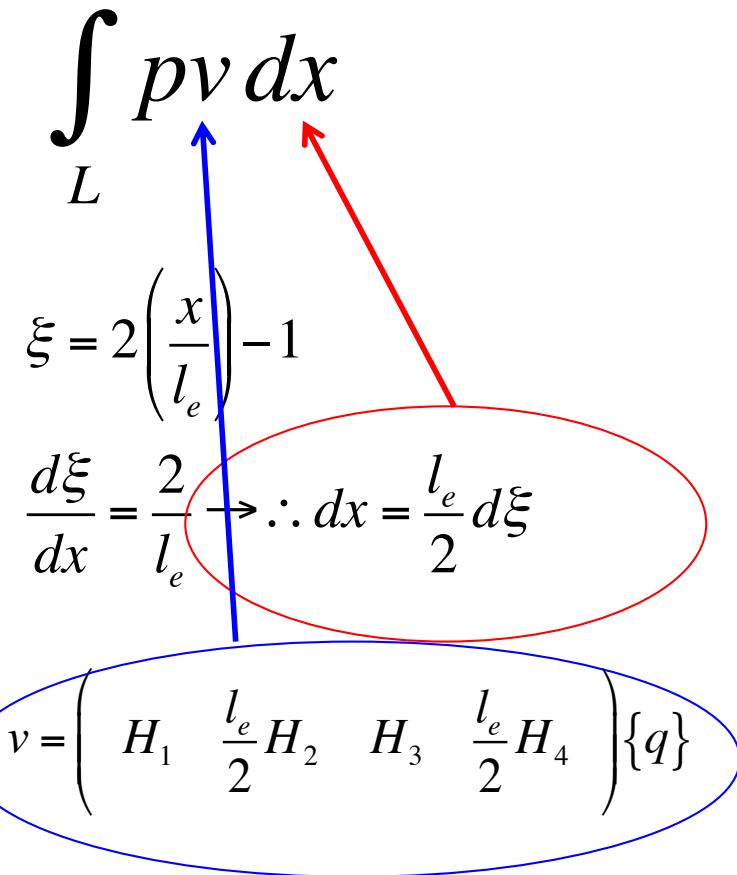
$$U_e = \frac{1}{2} EI \int \left(\frac{d^2 v}{dx_2} \right)^2 dx$$

Beam Element



$$\begin{aligned}
 & \int_L p v \, dx \\
 &= \int_{-1}^1 p \begin{pmatrix} H_1 & \frac{l_e}{2}H_2 & H_3 & \frac{l_e}{2}H_4 \end{pmatrix} \{q\} \frac{l_e}{2} d\xi \\
 &= \frac{pl_e}{2} \boxed{\int_{-1}^1 \begin{pmatrix} H_1 & \frac{l_e}{2}H_2 & H_3 & \frac{l_e}{2}H_4 \end{pmatrix} d\xi} \{q\}
 \end{aligned}$$

Load vector: Distributed load, p



Beam Element

$$H_1 = \left(\frac{\xi^3 - 3\xi + 2}{4} \right)$$

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$$H_4 = \left(\frac{\xi^3 + \xi^2 - \xi - 1}{4} \right)$$



$$\int_{-1}^1 H_1 d\xi = \int_{-1}^1 \left(\frac{\xi^3 - 3\xi + 2}{4} \right) d\xi = 1$$

$$\int_{-1}^1 H_2 d\xi = \int_{-1}^1 \left(\frac{\xi^3 - \xi^2 - \xi + 1}{4} \right) d\xi = \frac{1}{3}$$

$$\int_{-1}^1 H_3 d\xi = \int_{-1}^1 \left(\frac{-\xi^3 + 3\xi + 2}{4} \right) d\xi = 1$$

$$\int_{-1}^1 H_4 d\xi = \int_{-1}^1 \left(\frac{\xi^3 + \xi^2 - \xi - 1}{4} \right) d\xi = -\frac{1}{3}$$

Beam Element

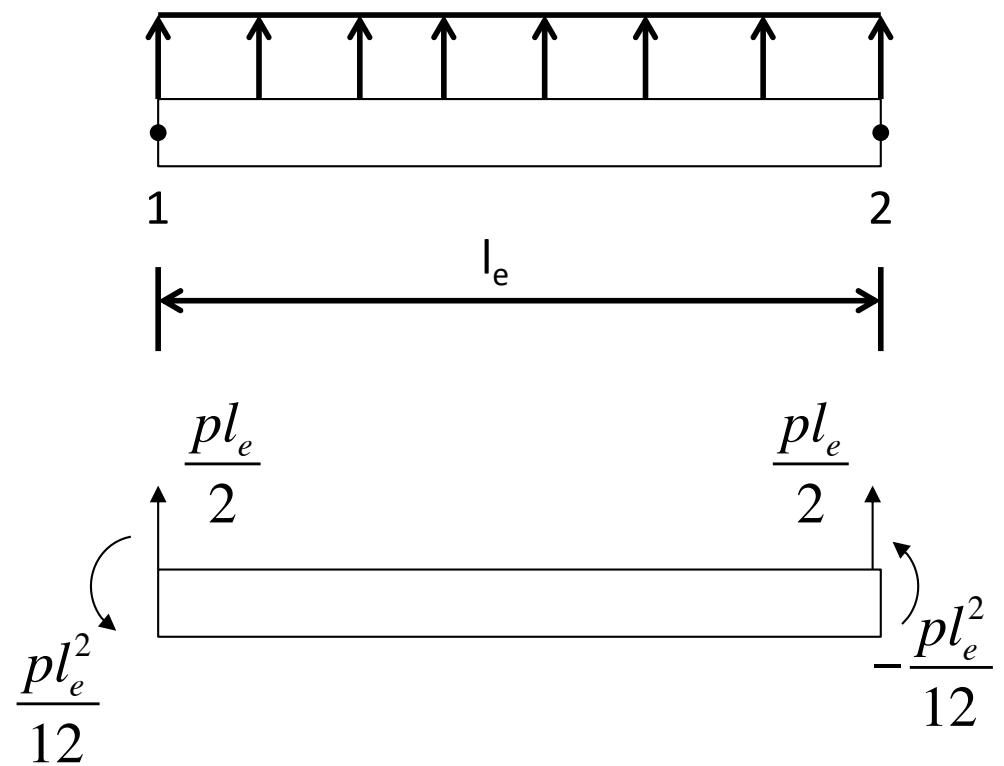
$$\begin{aligned}
 & \int_L p v \, dx \\
 &= \int_{-1}^1 p \begin{pmatrix} H_1 & \frac{l_e}{2} H_2 & H_3 & \frac{l_e}{2} H_4 \end{pmatrix} \{q\} \frac{l_e}{2} d\xi \\
 &= \frac{pl_e}{2} \begin{pmatrix} 1 & \frac{l_e}{6} & 1 & -\frac{l_e}{6} \end{pmatrix} \{q\} \\
 &= \begin{pmatrix} \frac{pl_e}{2} & \frac{pl_e^2}{12} & \frac{pl_e}{2} & -\frac{pl_e^2}{12} \end{pmatrix} \{q\} \\
 &= \{f_e\} \{q\}
 \end{aligned}$$

Load vector: Distributed load, p

$$\int_L p v \, dx$$

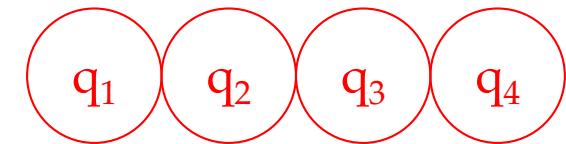
$$\boxed{\{f_e\} = \left(\begin{array}{cccc} \frac{pl_e}{2} & \frac{pl_e^2}{12} & \frac{pl_e}{2} & -\frac{pl_e^2}{12} \end{array} \right)}$$

Beam Element



Load vector: Distributed load, p

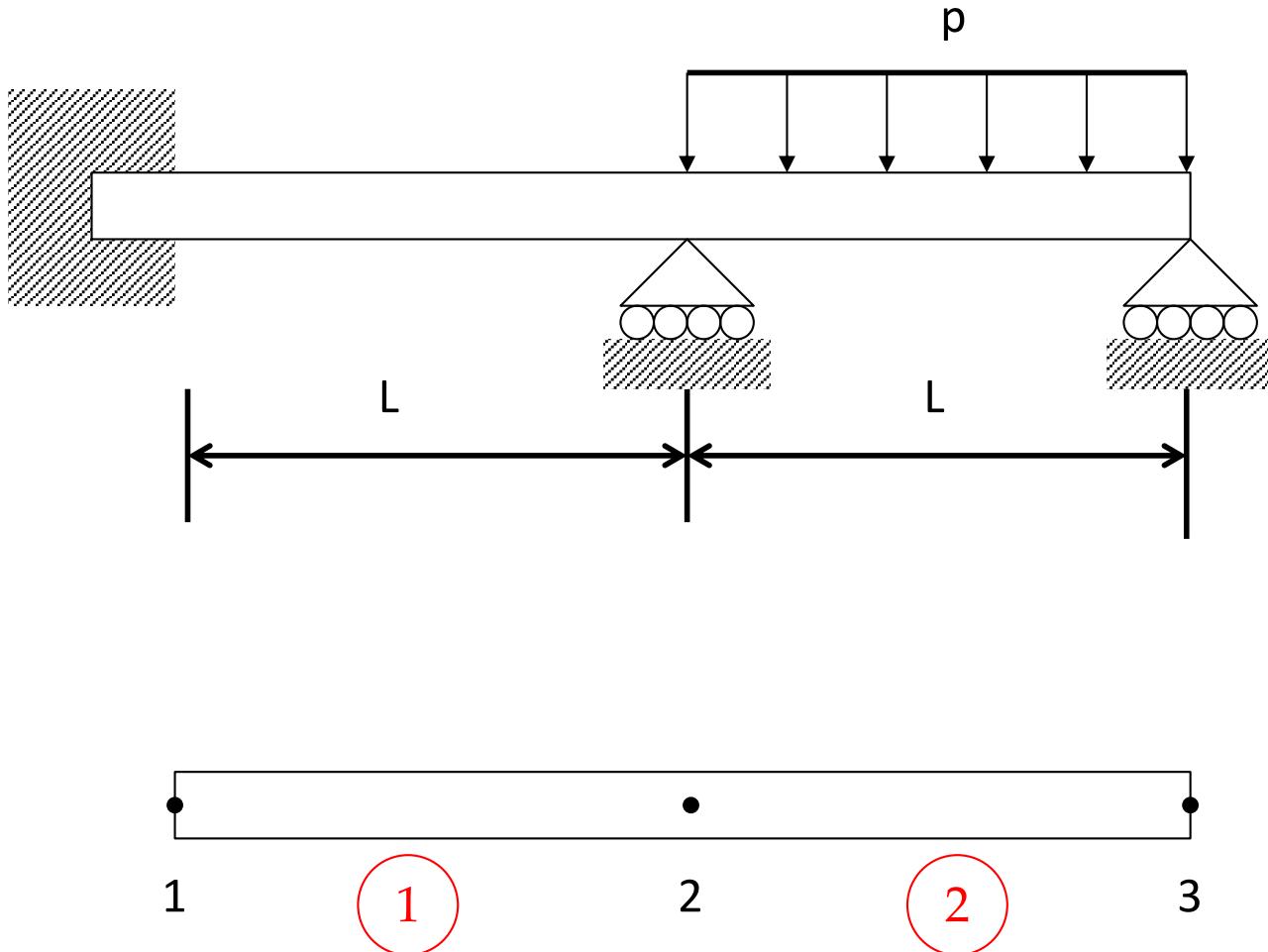
$$\int_L p v \, dx$$



$$\{f_e\} = \left(\begin{array}{cccc} \frac{pl_e}{2} & \frac{pl_e^2}{12} & \frac{pl_e}{2} & -\frac{pl_e^2}{12} \end{array} \right)$$

Assembly of Global Stiffness Matrix

Global Stiffness Matrix

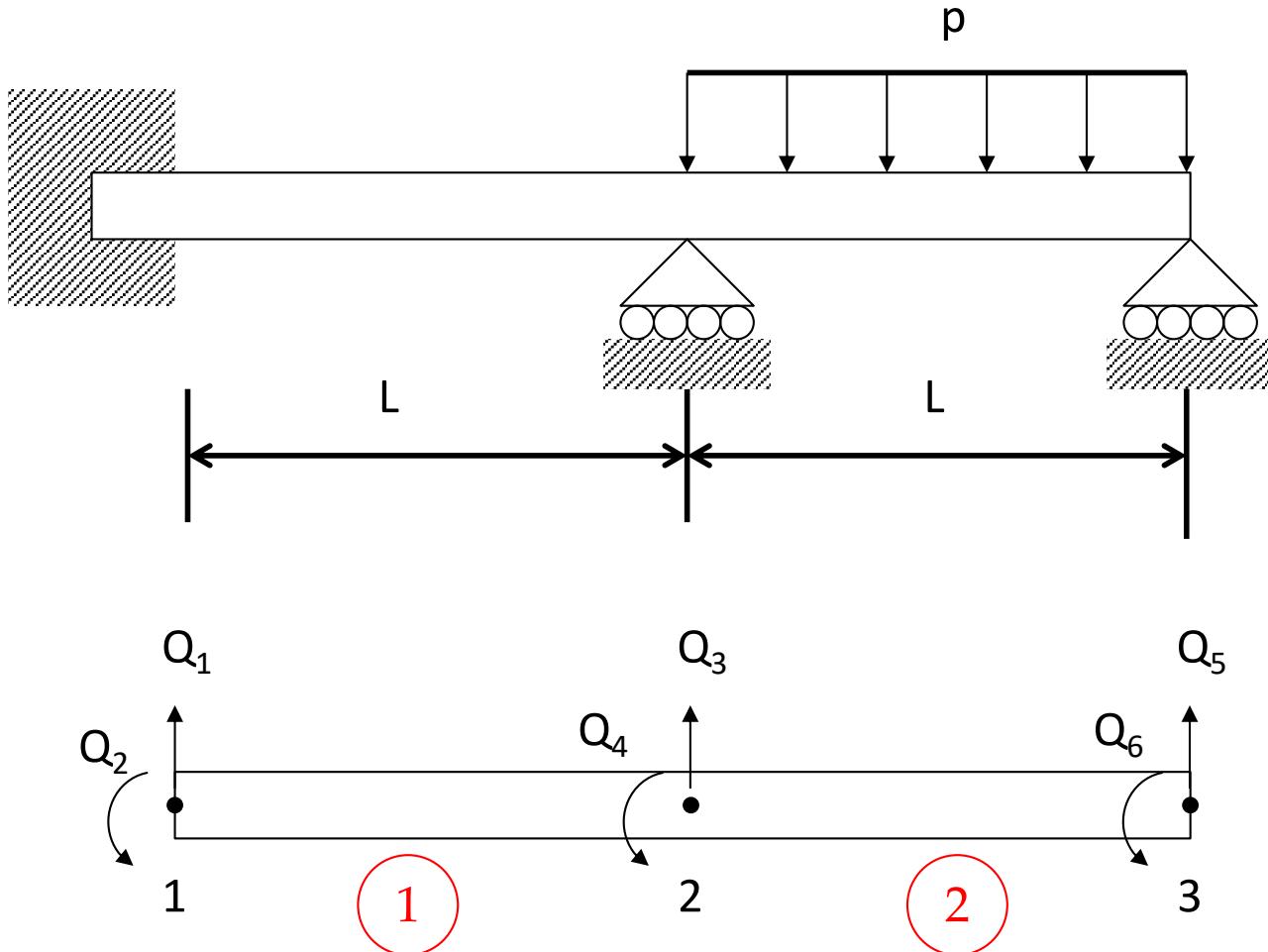


Assemble the global stiffness for the beam structure given using 2 beam elements.

Element Connectivity Table

Element	Nodes	
1	1	2
2	2	3

Global Stiffness Matrix



Assemble the global stiffness for the beam structure given using 2 beam elements.

Element Connectivity Table

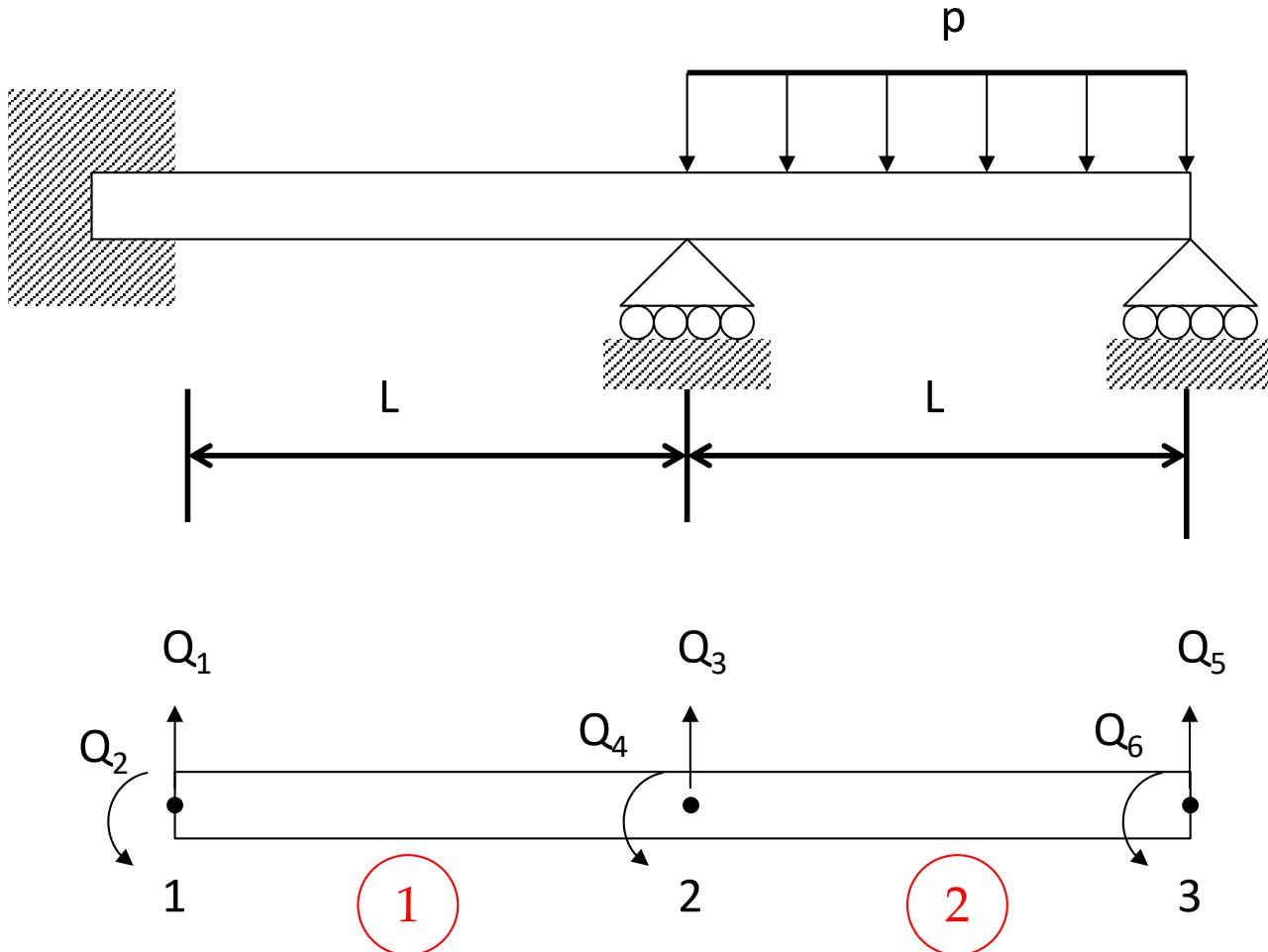
Element	Nodes	
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Global Stiffness Matrix

- Problem consists of 3 nodes, each with 2 degrees of freedom
- Therefore, stiffness matrix size is 6X6

$$K = \frac{EI}{L^3} \begin{pmatrix} 1 & 2 & 3 & 4 & 5 & 6 \\ k_{11} & k_{12} & k_{13} & k_{14} & k_{15} & k_{16} \\ k_{21} & k_{22} & k_{23} & k_{24} & k_{25} & k_{26} \\ k_{31} & k_{32} & k_{33} & k_{34} & k_{35} & k_{36} \\ k_{41} & k_{42} & k_{43} & k_{44} & k_{45} & k_{46} \\ k_{51} & k_{52} & k_{53} & k_{54} & k_{55} & k_{56} \\ k_{61} & k_{62} & k_{63} & k_{64} & k_{65} & k_{66} \end{pmatrix} \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \\ 5 \\ 6 \end{matrix}$$

Global Stiffness Matrix



Assemble the global stiffness for the beam structure given using 2 beam elements.

Given:

$$k_e = \frac{EI}{l_e^3} \begin{pmatrix} 12 & 6l_e & -12 & 6l_e \\ 6l_e & 4l_e^2 & -6l_e & 2l_e^2 \\ -12 & -6l_e & 12 & -6l_e \\ 6l_e & 2l_e^2 & -6l_e & 4l_e^2 \end{pmatrix}$$

Global Stiffness Matrix

1 2 3 4

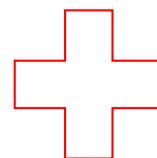
$$k_1 = \frac{EI}{L^3} \begin{pmatrix} k_{11} & k_{12} & k_{13} & k_{14} \\ k_{21} & k_{22} & k_{23} & k_{24} \\ k_{31} & k_{32} & k_{33} & k_{34} \\ k_{41} & k_{42} & k_{43} & k_{44} \end{pmatrix} \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \end{matrix}$$

$$K = \frac{EI}{L^3} \begin{pmatrix} 12 & 6L & -12 & 6L & k_{15} & k_{16} \\ 6L & 4L^2 & -6L & 2L^2 & k_{25} & k_{26} \\ -12 & -6L & 12 & -6L & k_{35} & k_{36} \\ 6L & 2L^2 & -6L & 4L^2 & k_{45} & k_{46} \\ k_{51} & k_{52} & k_{53} & k_{54} & k_{55} & k_{56} \\ k_{61} & k_{62} & k_{63} & k_{64} & k_{65} & k_{66} \end{pmatrix}$$

$$k_e = \frac{EI}{l_e^3} \begin{pmatrix} 12 & 6l_e & -12 & 6l_e \\ 6l_e & 4l_e^2 & -6l_e & 2l_e^2 \\ -12 & -6l_e & 12 & -6l_e \\ 6l_e & 2l_e^2 & -6l_e & 4l_e^2 \end{pmatrix}$$

Global Stiffness Matrix

$$k_1 = \frac{EI}{L^3} \begin{pmatrix} k_{11} & k_{12} & k_{13} & k_{14} \\ k_{21} & k_{22} & k_{23} & k_{24} \\ k_{31} & k_{32} & k_{33} & k_{34} \\ k_{41} & k_{42} & k_{43} & k_{44} \end{pmatrix} \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \end{matrix}$$



$$k_2 = \frac{EI}{L^3} \begin{pmatrix} k_{33} & k_{34} & k_{35} & k_{36} \\ k_{43} & k_{44} & k_{45} & k_{46} \\ k_{53} & k_{54} & k_{55} & k_{56} \\ k_{63} & k_{64} & k_{65} & k_{66} \end{pmatrix} \begin{matrix} 3 \\ 4 \\ 5 \\ 6 \end{matrix}$$

$$K = \frac{EI}{L^3} \begin{pmatrix} 12 & 6L & -12 & 6L & k_{15} & k_{16} \\ 6L & 4L^2 & -6L & 2L^2 & k_{25} & k_{26} \\ -12 & -6L & 12+12 & -6L+6L & -12 & 6L \\ 6L & 2L^2 & -6L+6L & 4L^2+4L^2 & k_{45} & 2L \\ k_{51} & k_{52} & -12 & -6L & 12 & -6L \\ k_{61} & k_{62} & 6L & 2L^2 & -6L & 4L^2 \end{pmatrix}$$

$$k_e = \frac{EI}{l_e^3} \begin{pmatrix} 12 & 6l_e & -12 & 6l_e \\ 6l_e & 4l_e^2 & -6l_e & 2l_e^2 \\ -12 & -6l_e & 12 & -6l_e \\ 6l_e & 2l_e^2 & -6l_e & 4l_e^2 \end{pmatrix}$$

Global Stiffness Matrix

$$K = \frac{EI}{L^3} \begin{pmatrix} 1 & 2 & 3 & 4 & 5 & 6 \\ 12 & 6L & -12 & 6L & 0 & 0 & 1 \\ 6L & 4L^2 & -6L & 2L^2 & 0 & 0 & 2 \\ -12 & -6L & 24 & 0 & -12 & 6L & 3 \\ 6L & 2L^2 & 0 & 8L^2 & -6L & 2L & 4 \\ 0 & 0 & -12 & -6L & 12 & -6L & 5 \\ 0 & 0 & 6L & 2L^2 & -6L & 4L^2 & 6 \end{pmatrix}$$

By the end of the notes:

You are expected to be able:

- To assemble the stiffness matrix and load vector for a beam element
- To assemble the global stiffness matrix for a beam structure